

## Activity Report during May 2014 to April 2019

by

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#### **Phase-change Thermal Systems Laboratory**





### In this presentation

- Introduction: Liquid-vapour/gas interfacial systems
- Engineering systems involving interfacial thermo-hydrodynamics
  - Pulsating Heat Pipe
  - Loop Heat Pipe
  - Spray Cooling of LEDs
  - Enrichment of Heavy Metals
  - Nuclear Containment Safety



Space and terrestrial sector (thermal management application)



Nuclear engineering sector (Safety and strategic application)

- Experimental techniques and representative results (HSV/ IRT/ PIV/ CFM/ XRT)
- Summary and Outlook









Taylor slug flows

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#### **Aims and Objectives**

Interface shape Interfacial heat and mass transfer Three-phase contact line dynamics Force interactions: surface, viscous, inertia, gravity Wall Transport: shear and thermal energy Interaction of interfaces **Multi-Scale Effects** Instrumentation Scaling laws Instabilities

#### **IRT: Porous media**



Sprays/Jets/Mist





High speed videography: deforming and merging interfaces



**Pool Boiling** 



**Confocal interface** microscopy



Dropwise Condensation

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#### **Experimental research: Challenges**

- Control on the boundary conditions: Heat flux/ Temperature/ Wall
- Visualization, coupled with application of boundary conditions
- Strong surface effects: repeatability of experimental data
- Instrumentation at microscale: Intrusive vs non-intrusive
- Viability and applicability of assumptions
- Purity of materials/ dissolution of gases
- Optical alignment/ Signal to noise ratio
- Thermal-hydrodynamic coupling
- Thermal conjugate effects
- Vacuum and leakage



GOAL

Local level understanding to global system development



Work undertaken on Engineering Systems with Strong Involvement of Interfacial Physics





commence

## **Pulsating Heat Pipe**

Simple meandering capillary tube
No wick or porous structure inside it
Evacuated, filled partially with a fluid



Glass tube PHP Video

Aluminum plate PHP

Evaporator

Condenser



#### **Loop Heat Pipe**



- Highly efficient mono-porous/ bi-porous wick structure  $\mathbf{O}$
- Excellent passive design for high heat removal  $\mathbf{O}$
- Invented by Dr. Yuri Maydanik in Russia  $\mathbf{O}$





#### **Spray Impingement Cooling**



#### Spray Cooling



#### **Enrichment of Heavy Metals**







- (a) Schematic of the reflux condensation experiment
- (b) A conical shaped reflux condensation chamber used for condensation of Bismuth
- (c) Typical condensation patterns of Bismuth on the substrate at 400°C and 20° inclination angle (Experiments: BARC, Mumbai, India)



Cooling of containment walls of nuclear reactor



#### **Reactor Containment Safety**



#### **THYCON Facility – IIT Kanpur**

- Experimentally simulating post-severe accident scenario
- Steam condensation in the presence NCGs
- Steam + Air + Hydrogen

0 = +90'

 $\Theta = +87$ 

Sessile Mode

 $\Theta = +45$ 

 $\Theta = 0^{\circ}$ 

Vertical Surface Pendant Mode

Sumple or DV D



## **Several Other Applications**



#### **Integrated electronics cooling**



Gas-liquid micro-reactors

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#### **Transport in fuel cells**



#### **Microfluidic devices**

- Compact
- High area/volume
- Better transport



edrawn from: Brivio, M., Verboom, W., & Reinhoudt, D. N. (2006). Miniaturized continuous flow saction vessels: influence on chemical reactions. Lab on a Chip, 6, p. 329.





## **Scaling of Forces**

- Dominance of interfacial force
  - Bond number (Bo) Gravity/surface tension

$$Bo = \frac{\Delta \rho dD^2}{\sigma} < 2$$

#### Surface tension >> gravity



#### Droplet motion/ coalescence

#### Meniscus shape

- · Young-Laplace at equilibrium
- Capillary number (Ca) Viscous/surface tension

$$Ca = \frac{\mu V}{\sigma} < 10^{-3}$$

#### Surface tension >> Viscous

#### • Inertia

• Weber number (We) – Inertia/surface tension

$$We = \frac{\rho U^2 D}{\sigma}$$

Surface tension >> Inertia



#### Moving contact lines



Bubble growth



## Interfacial Transport: Multi-scale Hierarchical System

#### Stage 1 (Atomic to Nanoscale)

→ Molecular potentials, Adatom dynamics, Cluster dynamics, surface diffusion, Stable cluster size and population density, Accommodation coefficient

#### Stage 2 (Nanoscale to Microscale)

→ Film stability, topography interaction, stable interfaces, pinning dynamics, wetting-dewetting dynamics, Young-Laplace condition

#### Stage 3 (Microscale to Macroscale)

→ Interfacial growth, coalescence, merger, interaction of surface force, body force, viscous force and inertia force, momentum flux transfer



#### **Experimental Tools: Fluid-Thermal Laboratory**



Laser confocal microscope



Thermal diffusivity system



X-Ray tomography



Infra-red camera



**High-speed camera** 



**Micro PIV** 



Goniometer



## High Speed Videography



#### **Two-phase Flow and Heat Transfer**



Upward flow boiling patterns in a 2.0 mm tube under different input heat flux conditions



 $J_{tot} = 0.15 \text{ m/s}$ 



Effect of surface morphology on spray impingement





# Sliding path of moving droplet in experiment and simulation







from simulation



#### **Bubble Growth in Binary Mixtures of Aqueous Ethanol**

2.0% ethanol T<sub>sat</sub> = 50°C, q" = 0.046 MW/m<sup>2</sup>







25.0% ethanol T<sub>sat</sub> = 50°C, q" = 0.046 MW/m<sup>2</sup>

#### Effect of surface roughness

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R<sub>a</sub> = 20 μm



## Infra Red Thermography

#### **IRT of Micro-channel Flows and Droplets**



Flow patterns in a PHP



#### **Transient Temperature Profiles and Nusselt number**



J<sub>tot</sub> = 0.11 m/s





Variation of wall and fluid temperature with time for Taylor bubble train flow for  $\beta$  = 0.384 and 0.652, respectively.



Axial variation of Nusselt number for different volume flow ratio of Taylor bubble-train flow



#### Loop Heat Pipe: IRT for Wick Design



(a) Schematic of loop heat pipe (b) Cross-section of evaporator and unit cell (c) Infrared imaging setup (d) Location of the evaporation front from thermography

System level thermography



## Particle Image Velocimetry



## **PIV of single meniscus**



interface for  $U_{avg} = 0.166$  mm/s, (Ca = 2.27e-6):



- Enhancement in transport due to V comp.
- Away from interface, U is parabolic
- Close to the interface U-velocity reduces,
- Flow becomes 3D very near to interface
- Circulating vortices are observed behind the interface

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#### Flow field and its modeling



(a) Streamlines of water plug at Ca = 1e-3 (b) Meniscus shape for various capillary tube wettability at commencement of motion (contact angle 140° for avg. velocity ( $U_{avg}$ ) is = 0.038 mm/s (Ca = 5e-4))

Variation of Poiseuille number ( $C_{f}$ ·Re) experienced along the wall due to the steady meniscus motion, for the three cases of wettability respectively.



## **PIV of Oscillating Taylor Plug**







## **Confocal Microscopy**



#### **Change of Mesh Wettability through Heat Treatment**





untreated SS#100 mesh



8µl drop on untreated SS#100 mesh CA ~ 120°









Heat treated SS#200 mesh

Through thermal oxidation, the SS Mesh is made hydrophilic. 31

SS#200



untreated SS#200 mesh 8μl drop on untreated SS#200 mesh CA ~ 120°

The SS Mesh is inherently hydrophobic by nature.



#### **Microstructure Growth on Heat Treatment**



In untreated mesh: only primary pores are present

Mesh #100: Average pore size 148 µm Wire diameter = 94 µm Mesh #200: Average pore size 76 µm Wire diameter = 47 µm

> Metal oxide layers are usually hydrophilic and moreover, oxide structures provide secondary micro-pores Chemical+Physical

# In heat treated mesh: primary as well as secondary pores due to oxide growth Wire diameter tends to increase/swelling (8-12 μm) Consequently, two length scales appear



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## **Visualization of Thin-film Evaporation** through Confocal Microscopy



Time evolution of menisci during evaporation in saturated screen mesh

**Evaporation dynamics ?** Microscale fluid flow during evaporation ?



Fluid motion during thin film evaporation in saturated screen mesh



## Visualization of Thin-film Evaporation through Confocal Microscopy



Untreated SS#100 mesh

No liquid film over the wires of mesh Contact line motion on the wires Higher meniscus RoC at rupture Lesser time for complete evaporation



#### HT SS#100 mesh

Liquid film over the wires – secondary pore No CL motion – secondary pore –film hold up Lower meniscus RoC at rupture Longer time for complete evaporation



### **Evaporation Mechanism**



Hydrophobic nature of untreated meshes – CL motion – No liquid at the wires Larger pore spacing in untreated meshes – High meniscus RoC at ruptures Untreated meshes take lower time to evaporate than HT mesh HT meshes – completely wetting – secondary pores – increased pore saturation



## Summary and Outlook



### Summary and Outlook

- Fluid-fluid and Fluid solid interfaces are ubiquitous in engineering systems
- Discerning thermo-hydrodynamics of interfaces poses challenging problems
- Local level transport is intrinsically linked with the system level performance
- Multiple-scales/physics interact manifesting a hierarchical problem definition (nano → micro → macro)
- To be meaningful, experiments require strict control of boundary conditions
- Several probing tools → effective exploitation needed to discern local physics
- Interdisciplinary skills need to be groomed in students  $\rightarrow$  cooperation/sharing
- Interesting transport physics awaits exploration and translation into products!