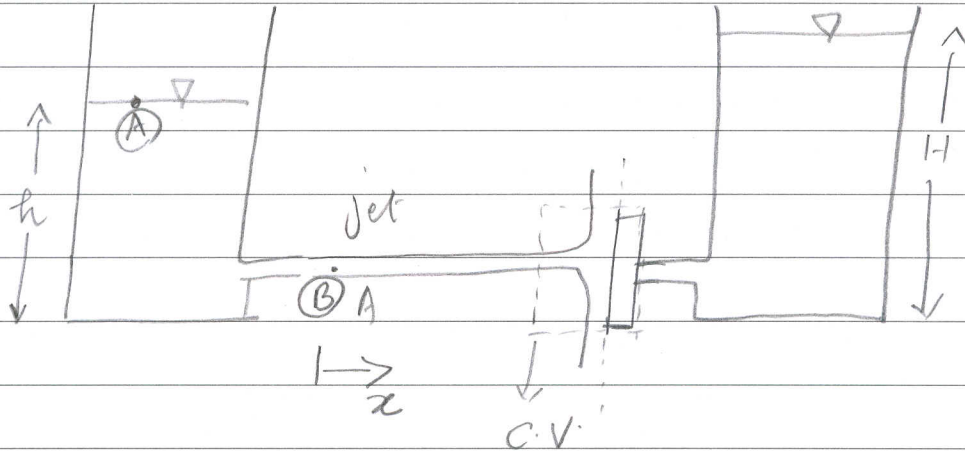


①



(Steady) Integral momentum balance:

$$F = \int_{C.S.} \rho \mathbf{u} \cdot \mathbf{n} dA \quad \text{-----} \quad \boxed{2 \text{ points}}$$

x-component: (uniform flow)

$$F_x = \int_{C.S.} \rho v_x (-v_x) dA$$

$$F_x = -\rho v_x^2 A$$

$$v_x = v_{jet}$$

$$\boxed{2 \text{ points}} \quad -\rho g H A = -\rho v_{jet}^2 A \quad \text{---} \quad \text{①}$$

$$F_x = -\rho g H A$$

By applying Bernoulli eqn between (A) and (B)

$$\Rightarrow \frac{P_A}{\rho} + \frac{v_A^2}{2} + g z_A = \frac{P_B}{\rho} + \frac{v_B^2}{2} + g z_B$$

$$P_A = P_B = P_{atm}; \quad v_A \approx 0 \quad z_A - z_B = h, \quad v_B = v_{jet}$$

①

\Rightarrow Bernoulli eqn gives

$$V_{jet}^2 = 2gh$$

2 points

Use this in Eq. ① of prev page:

$$gH = V_{jet}^2$$

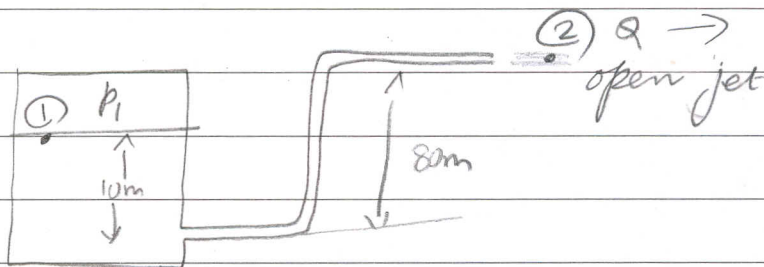
$$gH = 2gh$$

\Rightarrow

$$h = \frac{H}{2}$$

2 points

②



apply energy balance between ① and ②:

$$p_1 + \frac{\alpha_1 V_1^2}{2} + g z_1 = \frac{p_2}{\gamma} + \frac{\alpha_2 V_2^2}{2} + g z_2 + f \frac{L}{D} \frac{V_2^2}{2g} + \sum_i K_i \frac{V_2^2}{2g}$$

$$p_1 - p_2 = \rho g h ; V_1 \approx 0$$

$$V_2 = V_{pipe} \quad (\text{by mass balance})$$

$$V_1 \approx 0$$

$$z_1 = 10 \text{ m}$$

$$z_2 = 80 \text{ m}$$

2 points

$$\Rightarrow \frac{P_{ig}}{\rho} = \frac{V^2}{2} \left[\alpha_2 + f \frac{L}{D} + K_{ent} + 2 K_{90} \right] + g \cdot 70 \quad \boxed{2 \text{ points}}$$

$$Q = 60 \frac{\text{m}^3}{\text{hr}} = \frac{60}{3600} \frac{\text{m}^3}{\text{s}} = \frac{1}{60} \frac{\text{m}^3}{\text{s}} \quad \boxed{2 \text{ points}}$$

$$V = \frac{Q}{A} = \frac{1}{60} \cdot \frac{4}{\pi \cdot 0.05^2} = \boxed{8.49 \text{ m/s}}$$

$$\Rightarrow \text{Re in the pipe} = \frac{8VD}{\mu} = \frac{10^3 \cdot 8.49 \cdot 0.05}{15^3} = 4.245 \times 10^5$$

\Rightarrow flow is turbulent \Rightarrow from f -Re chart, $f \approx 0.0136$ 2 points

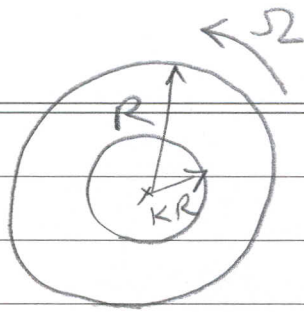
so

$$P_{ig} = \frac{\rho V^2}{2} \left[1 + \frac{0.0136 \cdot 170}{0.05} + 0.5 + 2 \times 0.95 \right] + \rho g 70$$

$$= 2.475 \times 10^6 \text{ Pa}$$

$$\boxed{P_{ig} = 2.475 \text{ MPa}} \quad \boxed{2 \text{ points}}$$

3



only non-zero velocity component: $v_\theta(r)$.

Steady, axisymmetric flow in θ -direction.

$v_\theta, p \Rightarrow$ indep of θ

$$\mu \frac{\partial}{\partial r} \left[\frac{1}{r} \frac{\partial}{\partial r} (r v_\theta) \right] = 0 \quad \boxed{2 \text{ points}}$$

$$\Rightarrow \frac{\partial}{\partial r} (r v_\theta) = C_1 r$$

$$r v_\theta = C_1 \frac{r^2}{2} + C_2$$

$$\text{or } \boxed{v_\theta = C_1 \frac{r}{2} + \frac{C_2}{r}} \quad \boxed{2 \text{ points}}$$

B.C $v_\theta(r=kR) = 0$

$v_\theta(r=R) = \Omega R$

$$v_\theta = \frac{\Omega R}{(1-k^2)} \left[\frac{r}{R} - \frac{k^2 R}{r} \right] \quad \boxed{2 \text{ points}}$$

$$\tau_{r\theta} = \mu r \frac{\partial}{\partial r} \left(\frac{v_\theta}{r} \right)$$

$$\tau_{r\theta} = \frac{\mu \Omega R}{\left(\frac{1}{k} - k\right)} \frac{2kR}{r^2}$$

Stress on the inner cylinder at $r = kR$

$$\tau_{r\theta} \Big|_{r=kR} = \frac{2\mu\Omega}{(1-k^2)} \quad \boxed{2 \text{ points}}$$

Torque at the cylinder ($r = kR$)

$$= \tau_{r\theta} \Big|_{r=kR} \times 2\pi kR L \times kR$$

$$\boxed{\begin{array}{l} \text{Torque} \\ \text{on the} \\ \text{inner cylinder} \end{array}} = \frac{4\pi\mu\Omega k^2 R^2 L}{(1-k^2)} \quad \boxed{2 \text{ points}}$$

_____ x _____ x _____
z

④ (a) Potential flow:

$$\phi = Ay^2 \cos 2\theta$$

$$v_r = \frac{\partial \phi}{\partial r} = 2Ar \cos 2\theta \quad \text{---} \quad \boxed{1 \text{ point}}$$

$$v_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = -2Ar \sin 2\theta \quad \text{---} \quad \boxed{1 \text{ point}}$$

$$v_\theta = -\frac{\partial \psi}{\partial r} ; \quad v_r = \frac{1}{r} \frac{\partial \psi}{\partial \theta}$$

$$\frac{1}{r} \frac{\partial \psi}{\partial \theta} = 2Ar \cos 2\theta$$

$$\Rightarrow \psi = Ar^2 \sin 2\theta + C_1(r) \quad - \quad \boxed{2 \text{ point}}$$

$$\frac{\partial \psi}{\partial r} = 2Ar \sin 2\theta$$

$$\Rightarrow \psi = Ar^2 \sin 2\theta + C_2(\theta) \quad - \quad \boxed{2 \text{ point}}$$

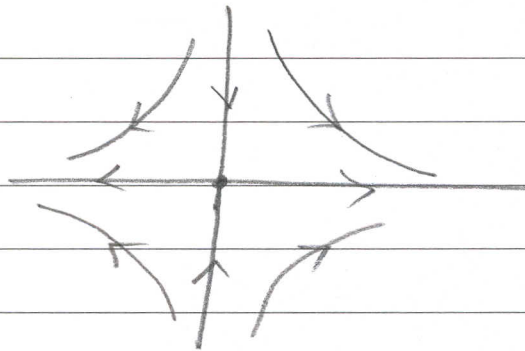
$$\Rightarrow \psi = Ar^2 \sin 2\theta$$

Consider the stream line $\psi = 0$.

$$\Rightarrow \sin 2\theta = 0$$

$$\Rightarrow 2\theta = n\pi$$

$$\theta = \frac{n\pi}{2}, \quad n = 0, 1, 2, \dots$$



$\boxed{4 \text{ points}}$

Flow near a
stagnation point

4(b)

$$C_f = \frac{0.664}{Re_x^{1/2}}$$

$$C_f = \frac{\tau_w}{\frac{1}{2} \rho U^2} = \frac{0.664}{\left(\frac{\rho U}{\mu}\right)^{1/2} x^{1/2}}$$

$$\tau_w(x) = \frac{\frac{1}{2} \rho U^2}{\left(\frac{\rho U}{\mu}\right)^{1/2} x^{1/2}} \cdot 0.664 \quad \text{--- [2 points]}$$

$$\text{Force on a single plate} = a \int_0^L \tau_w(x) dx$$

$$= \frac{1}{2} \rho U^2 a \left(\frac{\mu}{\rho U}\right)^{1/2} 0.664 \int_0^L \frac{dx}{x^{1/2}}$$

$$= 0.664 a \left(\frac{\mu}{\rho U}\right)^{1/2} \sqrt{L} \rho U^2$$

$$= 0.664 a \mu^{1/2} \rho^{1/2} U^{3/2} L^{1/2}$$

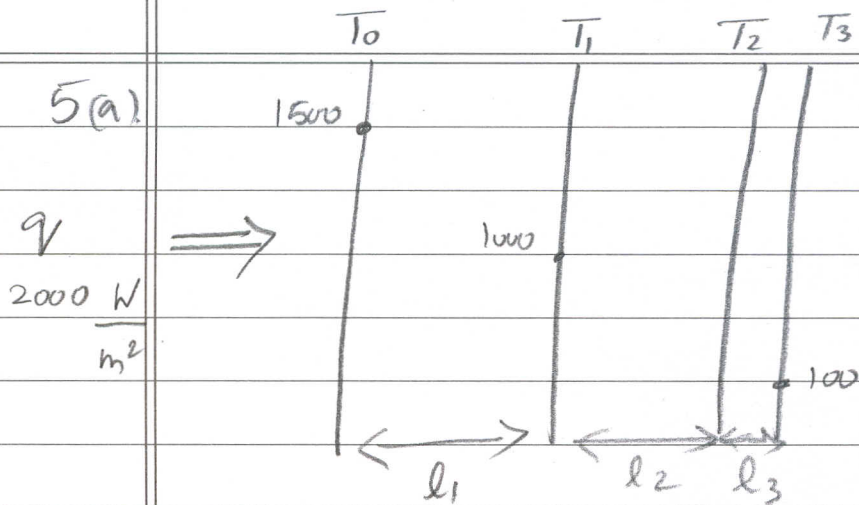
$$F \text{ on one plate} = 0.664 (\rho \mu L)^{1/2} a U^{3/2} \quad \text{--- [2 points]}$$

$$F \text{ on four plates} = 4 \cdot 0.664 (\rho \mu L)^{1/2} a U^{3/2}$$

By an integral momentum balance:

$$\Delta p = \frac{F}{a^2} = \frac{4 \times 0.664 (\rho \mu L)^{1/2} a U^{3/2}}{a^2}$$

$$\Delta p = 6.17 \text{ Pa} \quad \text{--- [4 points]}$$



$$q = k_1 \frac{\Delta T}{l_1} \quad \text{--- [2 points]}$$

$$\Rightarrow l_1 = \frac{k_1 (T_0 - T_1)}{q}$$

$$l_1 = \frac{4 \times 500}{2000} \Rightarrow \boxed{l_1 = 1 \text{ m}} \quad \text{--- [2 points]}$$

$$q = \frac{T_1 - T_3}{\left[\frac{l_2}{k_2} + \frac{l_3}{k_3} \right]}$$

$$2000 = \frac{900}{\left[\frac{l_2}{2} + \frac{0.01}{50} \right]}$$

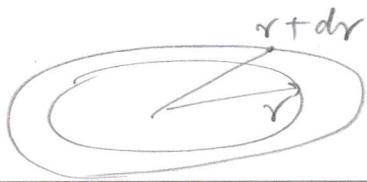
$$\Rightarrow \frac{l_2}{2} + \frac{0.01}{50} = \frac{900}{2000}$$

$$\Rightarrow \boxed{l_2 = 0.8996 \text{ m}} \quad \text{--- [2 points]}$$

$$2000 = \frac{T_2 - T_3}{\frac{0.01}{50}}$$

$\swarrow 100^\circ\text{C}$

$$\Rightarrow \boxed{T_2 = 100.4^\circ\text{C}} \quad \text{--- [2 points]}$$



5(b) heat in at $r \Rightarrow q_r (2\pi r) (2B) \Big|_r \rightarrow \boxed{2 \text{ points}}$

heat out at $r+dr \Rightarrow q_r (2\pi r) (2B) \Big|_{r+dr} \rightarrow \boxed{2 \text{ points}}$

heat lost by convection $\Rightarrow 2 \times (2\pi r) dr h(T - T_a)$

at steady-state: $\left[q_r (2\pi r) (2B) \right]_r - \left[q_r (2\pi r) (2B) \right]_{r+dr} - 2 \times 2\pi r dr h(T - T_a) = 0$

\div by $dr \Rightarrow -\frac{d}{dr} (r q_r) - r \frac{h(T - T_a)}{B} = 0 \quad \boxed{2 \text{ points}}$

$q_r = -k \frac{dT}{dr} \Rightarrow k \frac{d}{dr} \left(r \frac{dT}{dr} \right) - r \frac{h(T - T_a)}{B} = 0$

$\Rightarrow \boxed{\frac{1}{r} \frac{d}{dr} \left(r \frac{dT}{dr} \right) - \frac{h}{kB} (T - T_a) = 0}$

B.C. $\left. \begin{array}{l} \text{at } r = R_o, T = T_w \\ \text{at } r = R_o + R_i, \frac{dT}{dr} = 0 \end{array} \right\} \boxed{2 \text{ points}}$

6(a)

Transient Conduction:

$$\Theta = \frac{T - T_0}{T_1 - T_0}; \quad \tilde{r} = \frac{r \alpha}{R^2}$$

$$\Rightarrow \frac{\partial \Theta}{\partial \tilde{t}} = \frac{1}{\tilde{r}^2} \frac{\partial}{\partial \tilde{r}} \left(\tilde{r}^2 \frac{\partial \Theta}{\partial \tilde{r}} \right)$$

IC: $\Theta = 0$ at $\tilde{t} = 0$

BC: $\Theta = 1$ at $\tilde{r} = 1$.

let $\Theta(\tilde{r} = 0, \tilde{t} = \tilde{t}) = 0.99$ 2 points

2 points \tilde{t} will be the same for different spheres.

2 points $\tau_1 \propto \frac{1}{R_1^2} = \tau_2 \propto \frac{1}{R_2^2}$

$$m = \frac{4}{3} \pi R^3 \rho$$

$$R \propto m^{1/3}$$

$$\frac{\tau_2}{\tau_1} = \frac{R_2^2}{R_1^2} = \left(\frac{m_2}{m_1} \right)^{2/3}$$

if $m_2 = 2m_1$

$$\tau_2 = \tau_1 \cdot 2^{2/3}$$

(or)

$$\tau_2 = \tau_1 \cdot 4^{1/3}$$

2 points

6(b)

$$D \frac{d^2 C_A}{dz^2} - k C_A = 0$$

2 points

$$\text{BC } z=0, C_A = C_{A0}$$

$$z=L, \left. \frac{dC_A}{dz} \right|_{z=L} = 0$$

2 points

$$\frac{d^2 C_A}{dz^2} - \frac{k}{D} C_A = 0$$

$$\text{let } \Gamma = \frac{C_A}{C_{A0}}; \tilde{z} = \frac{z}{L}$$

$$\Rightarrow \frac{d^2 \Gamma}{d\tilde{z}^2} - \phi^2 \Gamma = 0 \quad \phi^2 = \frac{L^2 k}{D}$$

$$\Gamma = C_1 e^{\phi \tilde{z}} + C_2 e^{-\phi \tilde{z}}$$

2 points

$$\text{BC } \Gamma(\tilde{z}=0) = 1$$

$$\left. \frac{d\Gamma}{d\tilde{z}} \right|_{\tilde{z}=1} = 0$$

$$\Gamma(\tilde{z}) = \frac{e^{-\phi}}{e^{\phi} + e^{-\phi}} \left[e^{\phi \tilde{z}} - e^{-\phi \tilde{z}} \right] + e^{-\phi \tilde{z}}$$

2 points