

1. (a). Apply energy eqn. between (1) and (2).

$$\frac{p_1}{\rho g} + z_1 + \frac{v_1^2}{2g} = \frac{p_2}{\rho g} + z_2 + \frac{v_2^2}{2g} + h_L$$

$$p_1 = p_2 \approx P_{atm}; \quad v_1 = v_2 \approx 0.$$

$$\Rightarrow (z_1 - z_2) = h_L = 5.4 \frac{V_{tube}^2}{2g}$$

$$15m = \frac{5.4 \times V_t^2}{2 \times 9.8}$$

Ans part (a). $\Rightarrow V_t = 7.38 \text{ m/s.}$ [2 points] pump head

(b) $\frac{p_1}{\rho g} + \frac{v_1^2}{2g} + z_1 = \frac{p_2}{\rho g} + \frac{v_2^2}{2g} + z_2 - h_p + h_L$

$$(z_1 - z_2) = \frac{5.4 V_t^2}{2 \times 9.8} - h_p$$

D_b
 $V_t = 10 \text{ m/s} \Rightarrow h_p = 12.55 \text{ m.}$

$\frac{w_p}{g}$
 rate at which pump does work on the fluid
 $w_s = \frac{\text{rate at which pump does work on the fluid}}{\text{mass flow rate}}$

Power reqmt. of pump = Rate at which pump does work on the fluid

$$= \dot{m} \times w_p$$

$$= \dot{m} \times h_p \times g$$

$$= \rho Q \times h_p \times g$$

$$= 10^3 \times \frac{\pi}{4} (0.05)^2 \times 10 \times 12.55 \times 9.8$$

Rate at which pump does work on the fluid:

$$= \boxed{2415 \text{ Joules/sec}} \quad \text{Ans Part B. [2 points]}$$

(c) (d)

Flow is now reversed:

$$z_2 = z_1 - h_p + h_e$$

$$h_p = h_e + (z_1 - z_2)$$

$$= \frac{5.4 \times 10^2}{2 \times 9.8} + 15 = 42.55 \text{ m.}$$

$$w_p = h_p g ; \quad \dot{W}_p = \dot{m} w_p = \dot{m} h_p g \quad \text{[2 points]}$$

$$= \rho Q h_p g$$

Ans Part C.

(2)


$$= \boxed{8187 \text{ J/s}}$$

(d). Bernoulli between (1) & (2)
results in

$$g z_1 = g z_2 \Rightarrow \text{a contradiction since } z_1 \neq z_2.$$

The reason for this contradiction is that Bernoulli eqn. neglects all losses; which is not true here. The gravitational potential energy is converted to internal energy by viscous losses h_e , which is set to zero ~~line~~ in Bernoulli eqn. [2 points].

(2) (a) $\underline{v} \times d\underline{x} = 0$ along a streamline.


$$\Rightarrow \begin{vmatrix} \underline{i} & \underline{j} & \underline{k} \\ dx & dy & dz \\ u & v & w \end{vmatrix} = 0.$$

$$\Rightarrow \boxed{\frac{dx}{u} = \frac{dy}{v} = \frac{dz}{w}} \quad [2 \text{ points}].$$

(b) 2-D flow: $u = \frac{\partial \psi}{\partial y}$; $v = -\frac{\partial \psi}{\partial x}$.

$$\psi = \psi(x, y)$$

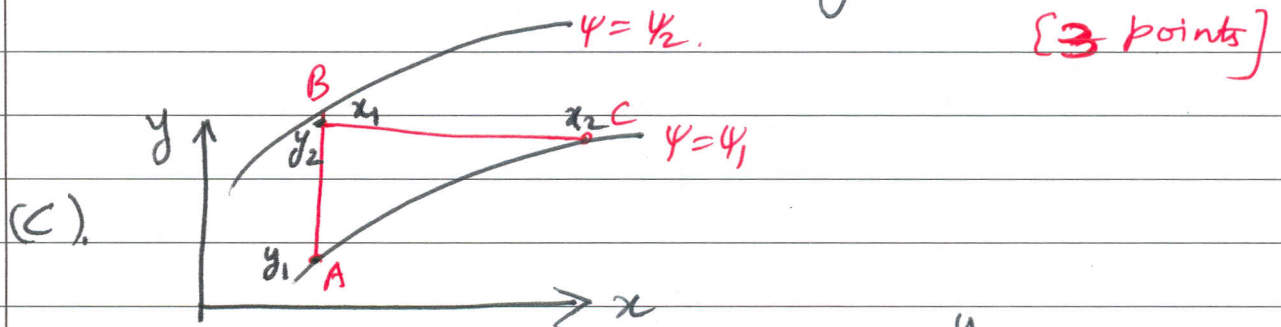
$$\Rightarrow d\psi = \frac{\partial \psi}{\partial x} dx + \frac{\partial \psi}{\partial y} dy.$$

$$\Rightarrow d\psi = -v dx + u dy$$

But $\frac{dx}{u} = \frac{dy}{v} \Rightarrow u dy - v dx = 0$

So $d\psi = 0$ along a streamline

$\Rightarrow \psi$ is a constant along a streamline. ()



along AB Q (per width) = $\int_{y_1}^{y_2} u dy$

$$= \int_{y_1}^{y_2} \frac{\partial \psi}{\partial y} dy$$

Since $x = \text{const}$ at AB

$$Q = \int_{y_1}^{y_2} d\psi$$

$$= \psi(y_2) - \psi(y_1) = \psi_2 - \psi_1.$$

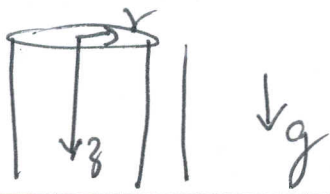
along BC $Q = \int_{x_1}^{x_2} u dx = - \int_{x_1}^{x_2} \frac{\partial \psi}{\partial x} dx = - \int_{x_1}^{x_2} d\psi$

$$= \psi(x_1) - \psi(x_2)$$

(4)

$$= \psi_2 - \psi_1$$

[2 points]



z.

Simplification of Continuity eqn. \Rightarrow [1 point]

(3)

(a)

$$v_z = v_z(r)$$

~~1 point~~

BC

$$p = p_{atm} \text{ at } r = aR$$

$$\tau_{rz} = \mu \frac{dv_z}{dr} = 0 \text{ at } r = aR$$

$$v_z(r) = 0 \text{ at } r = R$$

[1 point]

r-mom: $\frac{\partial p}{\partial r} = 0 \Rightarrow p = p(z)$ only

But $p = p_{atm}$ @ $r = aR \Rightarrow p = p_{atm}$

everywhere inside the liquid.

$$\Rightarrow \frac{\partial p}{\partial z} = 0$$

[1 point]

z-mom: $\mu \frac{1}{r} \frac{\partial}{\partial r} \left[r \frac{\partial v_z}{\partial r} \right] + \rho g = 0$ — [1 point]

$$\Rightarrow \frac{\partial}{\partial r} \left[r \frac{\partial v_z}{\partial r} \right] = \frac{-\rho g r}{\mu}$$

$$\Rightarrow v_z = -\frac{\rho g r^2}{4\mu} + C_1 \ln r + C_2$$

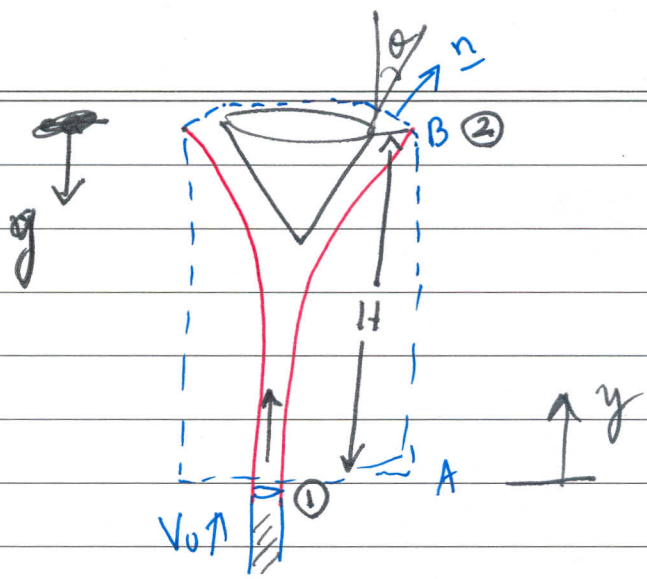
after using B.C's.

$$v_z = \frac{\rho g R^2}{4\mu} \left[1 - \frac{r^2}{R^2} + 2a^2 \ln \frac{r}{R} \right]$$
 — [1 point]

(b) $\tau_{rz} \Big|_{r=R} = \mu \frac{dv_z}{dr} \Big|_{r=R} = \frac{\rho g R}{2} (a^2 - 1)$ — [2 points]

(5)

④



Mass: $v_1 A_1 = v_2 A_2$. — 1 point

y-comp. Momentum: $F_g = \int_{CS} v_y \rho \underline{e} \cdot d\underline{A}$

2 points $Mg = [v_2 \cos \theta \rho v_2 A_2] - [v_1 \rho v_1 A_1]$

Bernoulli: $v_2^2 = v_1^2 - 2gH$. — 2 points

$v_1 = v_0$;

$\Rightarrow M = \frac{(v_0 - v_2 \cos \theta) \rho v_0 A_1}{g}$

$M = \left[\frac{v_0 - \sqrt{v_0^2 - 2gH} \cos \theta}{g} \right] \rho v_0 A_1$ 2 points

$H = 1\text{m}$
 $v_0 = 10\text{m/s}$

$D = 100\text{mm} = 0.1\text{m}$
 $\theta = 30^\circ$

$\Rightarrow M = 17.9\text{ kg}$. — 1 point

⑥