

USEFUL INFORMATION

(All required information is provided with the question paper)

- Density of water = 10^3 kg/m^3 , viscosity of water = 10^{-3} Pa s , $g = 9.8 \text{ m/s}^2$, $P_{atm} = 1.013 \times 10^5 \text{ Pa}$.
- Kinetic energy correction factor $\alpha = 2$ for laminar flow, $\alpha = 1$ for turbulent flow in pipes; Loss coefficient K for: entrance = 0.78, elbow (90°) = 0.9, elbow (45°) = 0.4; gate valve (fully open) = 0.3.
- Vector Identity: $(\mathbf{v} \cdot \nabla)\mathbf{v} = \nabla(\frac{1}{2}\mathbf{v} \cdot \mathbf{v}) + (\nabla \times \mathbf{v}) \times \mathbf{v}$

Navier-Stokes x -momentum equation in Cartesian coordinates:

$$\rho \left[\frac{\partial v_x}{\partial t} + v_x \frac{\partial v_x}{\partial x} + v_y \frac{\partial v_x}{\partial y} + v_z \frac{\partial v_x}{\partial z} \right] = -\frac{\partial p}{\partial x} + \mu \left[\frac{\partial^2 v_x}{\partial x^2} + \frac{\partial^2 v_x}{\partial y^2} + \frac{\partial^2 v_x}{\partial z^2} \right] + \rho g_x$$

Steady conduction equation in spherical coordinates:

$$\left[\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial T}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial T}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 T}{\partial \phi^2} \right] = 0$$

Steady diffusion equation in spherical coordinates:

$$\left[\frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial \rho_A}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial \rho_A}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 \rho_A}{\partial \phi^2} \right] = 0$$

Clearly state all the assumptions you make

1. A large tank containing water (shown in figure 1) has a small smoothly contoured orifice, from which a water jet exits with a velocity of V_{jet} . The height of water in the tank h as shown in the figure is 45.92 m. The diameter of the jet is 100 mm, and this jet moves horizontally to the right, where it is deflected by a cone that moves to the left at $V_{cone} = 14 \text{ m/s}$. The thickness t of the liquid film that leaves the cone at the radius $R (= 230 \text{ mm})$ is $t = 5.434 \text{ mm}$. Neglect effects of gravity for flow around the cone. Determine:
 - (a) The velocity V_{jet} of the liquid jet that comes out of the orifice, neglecting losses. [4 points]
 - (b) The velocity of the liquid that leaves the cone at $R = 230 \text{ mm}$. [4 points]
 - (c) The horizontal component of the external force on the cone required to maintain its motion. [6 points]

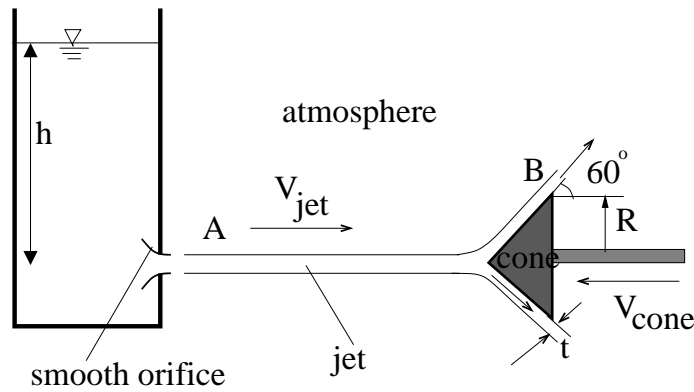


Figure 1: **Problem 1**

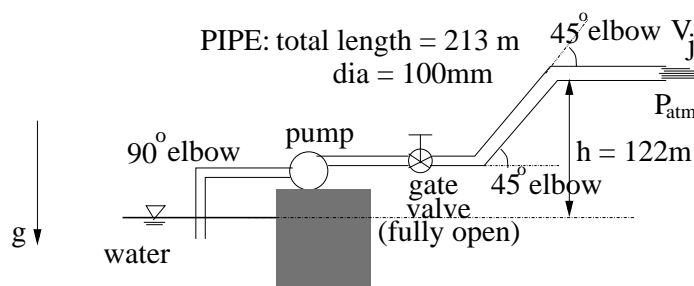


Figure 2: **Problem 2**

2. Water is to be pumped from a reservoir using a pipe system shown in figure 2. The flow rate in the pipe must be $0.038 \text{ m}^3/\text{s}$ and water must leave the exit of the pipe via a nozzle at a velocity $V_j = 37 \text{ m/s}$. The dimensions and fittings involved in the pipe system are indicated in the figure. The wall roughness factor $\epsilon = 0.0015 \text{ mm}$. Neglect losses at the exit nozzle. Determine the power input (in kW) to the pump required to achieve this flow. [12 points]

3. A belt of width W moves at a velocity V as shown in figure 3. A very viscous liquid fills the gap b between the belt and a stationary plate. Assume steady, fully-developed laminar flow between the belt and the plate, and do not neglect the effect of gravity. For this system:
 - (a) State the boundary conditions to solve the problem. [2 points]
 - (b) Derive the expression for velocity profile in the liquid [6 points]
 - (c) Derive the expression for volumetric flow rate of the liquid [2 points].
 - (d) If $\mu = 0.1 \text{ Pa s}$, $\rho = 10^3 \text{ kg/m}^3$, $b = 1 \text{ cm}$, $\theta = 45^\circ$, determine the velocity (in m/s) of the belt above which there is a net volumetric flow rate in the direction of the belt motion. [2 points]

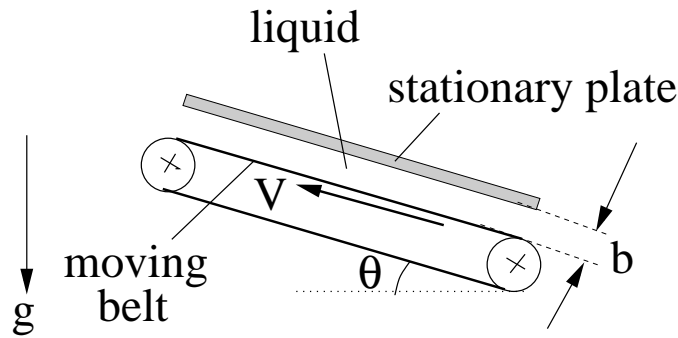


Figure 3: **Problem 3**

4. (a) Derive the Bernoulli equation for steady incompressible flow from the Euler equation for an inviscid fluid. Between which two points is the derived Bernoulli equation valid ? [7 points]
- (b) Prove that for a 2-D incompressible, irrotational flow the streamlines and equipotential lines are always orthogonal. [7 points]
5. (a) Consider a solid sphere of radius R_1 maintained at a temperature T_1 in a fluid at an ambient bulk temperature of T_0 (see figure 4 A). The convective heat transfer coefficient characterizing the heat loss at the boundary is h . This sphere is now wrapped with an insulating material of thermal conductivity k , such that the total radius becomes R_2 (figure 4 B).
 - i. Derive an expression for the thermal resistance of the insulating layer. [3 points]
 - ii. Is there a critical insulation thickness for this case ? If so, derive an expression for the critical insulation thickness. [4 points]

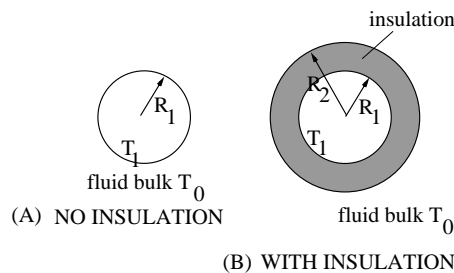
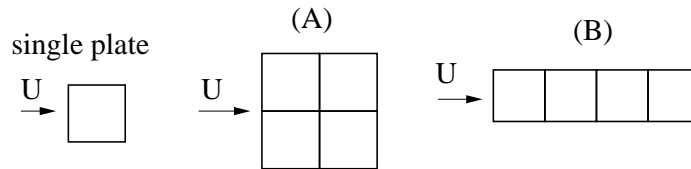


Figure 4: **Problem 5(a)**

- (b) A rice noodle in the shape of a long wire (1 mm diameter, $k = 1 \text{ W/(m K)}$, $C = 4000 \text{ J/(kg K)}$, $\rho = 2000 \text{ kg/m}^3$) of initial temperature 30°C is dipped in a vessel containing boiling water (100°C). The heat transfer coefficient for the noodle-water interface is $h = 1 \text{ W/(m}^2 \text{ K)}$. Calculate the Biot number for this case.

Based on the Biot number, what conclusion can you reach for the temperature distribution in the noodle ? Determine the time it takes for the center of the noodle to reach 80°C , when it is considered to be cooked. [7 points]

6. (a) Consider the steady laminar boundary layer flow past the top surface of square plate arrangements shown in figure 5. Compared to the friction drag on a single plate (area A , drag force F_1), how much larger is the drag on four plates together as in configurations (A) and (B) shown in figure 5. Will the drag on the two arrangements be the same ? Explain. [7 points]



TOP VIEW

Figure 5: **Problem 6(a)**

- (b) A naphthalene ball of diameter 1 cm is suspended in a large room with still pure air at 27°C and 1 atm. The surface temperature of naphthalene can be assumed to be 27°C and its vapor pressure at this temperature is 1 mm Hg. Assume naphthalene vapor to be an ideal gas. Determine the steady rate of evaporation of naphthalene (in kg/s). Assume that the radius of the naphthalene ball diameter remains constant in your calculation. Molecular weight of naphthalene is 128 kg/kmol; $R = 8314 \text{ m}^3 \text{ Pa}/(\text{kmol K})$; D for naphthalene in air is $5 \times 10^{-6} \text{ m}^2/\text{s}$. [7 points]
-